

MOHR'SCIRCLE CONCLUSIONS

1. Every plane passing through a point under bi-axis state of stress is represented by a radial line in the Mohr's circle.
2. XY coordinates of any point on the circle represent corresponding normal and shear stress on the oblique plane passing through that point, respectively.
3. Every plane passing through a point is represented in the Mohr's circle by double its actual angle (i.e. 2θ)
4. Centre of Mohr's circle always lies on X-axis & its coordinates are $\left[\frac{\sigma_1 + \sigma_2}{2}, 0 \right]$
5. Principal planes are represented by the radial lines lying on X-axis.
6. Principal stresses are equal to the X-coordinates of points of intersection of circle with the X-axis.
7. Maximum shear stress planes are represented by the radial lines which are parallel to Y-axis.
8. Normal stress on Max. shear stress plane (i.e. σ_{33}^*) is equal to X-coordinate of centre of Mohr's circle.
9. In-plane T_{max} is equal to radius of Mohr's circle.
p.e. In-plane $T_{max} = \text{Radius} = \pm \frac{(\sigma_1 - \sigma_2)}{2} = \pm \frac{1}{2} \sqrt{(\sigma_1 - \sigma_2)^2 + 4J^2}$
10. Planes of pure shear are represented by the radial lines which are passing through the points where circle intersects X-axis.
11. Shear stress on planes of pure shear (i.e. τ_{33}^*) is equal to the Y coordinate of points of intersection of circle with the Y-axis. (or) $\tau_{33}^* = \sqrt{-\sigma_1 \sigma_2}$
12. Planes of pure shear are possible when Mohr's circle intersects Y-axis [i.e. when $\sigma_1, 2$ are unlike in nature]

13. Resultant shear stress on any oblique plane is equal to length of the line which is joining the corresponding point on the circle with the origin.
14. Mohr's circle becomes tangential to Y-axis for uniaxial state of stress but never circle will be tangential to X-axis.
15. Reference plane may lie on X-axis or on Y-axis or in any one of the quadrants.
16. If Mohr's circle is given, stresses on principal planes, max-shear stress planes & planes of pure shear can be determined but plane location can't be determined.
17. If Mohr's circle intersects Y-axis then,
- principal stresses are unlike in nature. Hence, they are known as maximum tensile stress and maximum compressive stress.
 - planes of pure shear will exist.
 - In-plane $T_{max} = \text{Abs. } T_{max} = \text{Radius of Mohr's circle.}$
18. centre of Mohr's circle coincides with the origin under following conditions:-
- $\sigma_x = -\sigma_y$ and $\tau_{xy} = 0$
 - $\sigma_x = -\sigma_y$ and $\tau_{xy} \neq 0$
 - $\sigma_x = \sigma_y = 0$ and $\tau_{xy} = T$.
19. When centre of Mohr's circle with origin then,
- principal stresses are equal & unlike in nature.
(i.e. $\sigma_1/\sigma_2 = -1$)
 - planes of pure shear and maximum shear stress planes coincide with each other.
 - $(\sigma_{max})_{tensile} = (\sigma_{max})_{comp} = \text{In-plane } T_{max} = \text{Abs. } T_{max} = \tau_g^*$
 τ_g^* = Resultant stress on any O-P = Radius of circle.
20. Mohr's circle becomes a point on X-axis when $\sigma_x = \sigma_y$ & $\tau_{xy} = 0$. e.g.: - thin spherical pressure vessel under internal pressure [i.e. $\sigma_x = \sigma_y = \sigma_h = \frac{pd}{4t} + \tau_{xy} = 0$]